

# A Vibration Problem in Vertical Circulating Water Pumps

April 24, 2008

Presented by:

**Matthew A. Gaydon**

Bechtel Power Corporation

Frederick, MD 21703 USA

Tel: (301) 228-8360

[www.bechtel.com](http://www.bechtel.com)

# Vertical Circulating Water Pumps

- Two Pumps Operating in Parallel on Cooling Tower
- Single-Stage Mixed-Flow Type Pump
  - Semi-open Impeller Design
  - 77" Bowl Diameter / 54" Discharge
- Direct Drive with Induction Motor
  - 445 rpm (7.42 Hz)
  - 2,138 bhp
- Rated Condition:
  - 86,000 gpm
  - 86 feet



# Vibration Amplitude

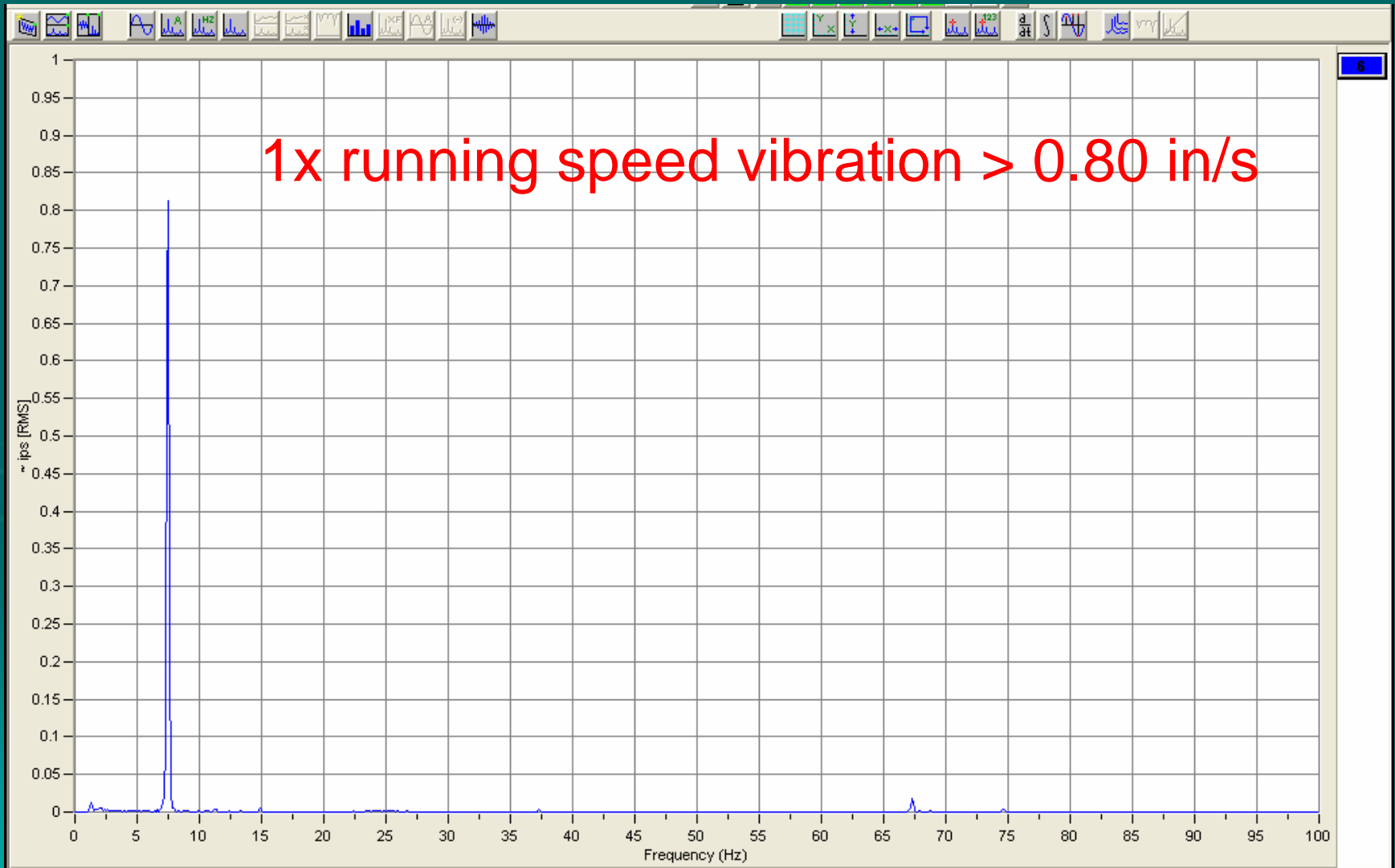
- Factory Test

- Acceptance: ISO 10816-3, Group 3, Flexible Support Class, Zone B/C (0.28 in/s RMS)
  - Top of Motor (parallel-to-discharge): 0.22 in/s RMS
  - Top of Motor (perpendicular-to-discharge): 0.16 in/s RMS

- Field Test

- Acceptance: ISO 10816-3, Group 3, Rigid Support Class, Zone B/C (0.18 in/s RMS)
  - Top of Motor (parallel-to-discharge): 0.13 in/s RMS
  - Top of Motor (perpendicular-to-discharge): 0.80 in/s RMS
  - Dominate frequency at 1X running speed.

# Initial Field Vibration Spectrum (perpendicular-to-discharge)



# Design-Phase FEA Results

- Foundation flexibility – Based on concrete properties from Civil Engineering design
- Motor reed critical frequency – Based on motor vendor information (+/- 15% margin)

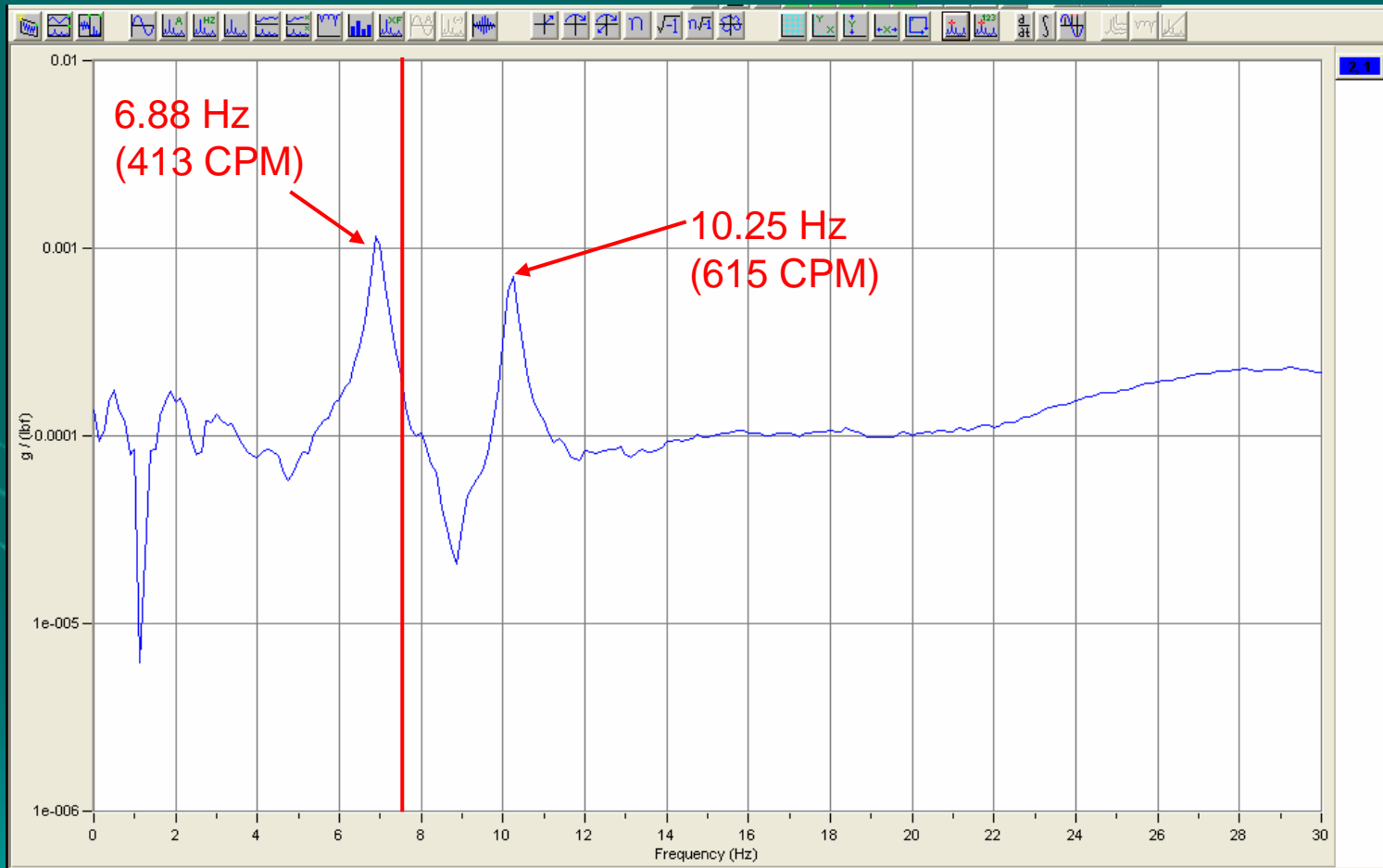
Running Speed = 7.42 Hz		
Mode	Natural Frequency (Hz)	Separation Margin
1 Parallel	4.70	-36.7%
1 Perpendicular	5.13	-30.8%
2 Parallel	8.32	12.1%
2 Perpendicular	8.73	17.7%

# Field Impact Test Conditions

- Pump installed per manufacturer's recommendations
- Pump not running and forebay filled with water
  - Accounts for partial entrained water mass
- Motor coupled and rotor lift set
  - Weight of pump rotor is supported at motor thrust bearing
- Discharge piping connected with flexible expansion joint

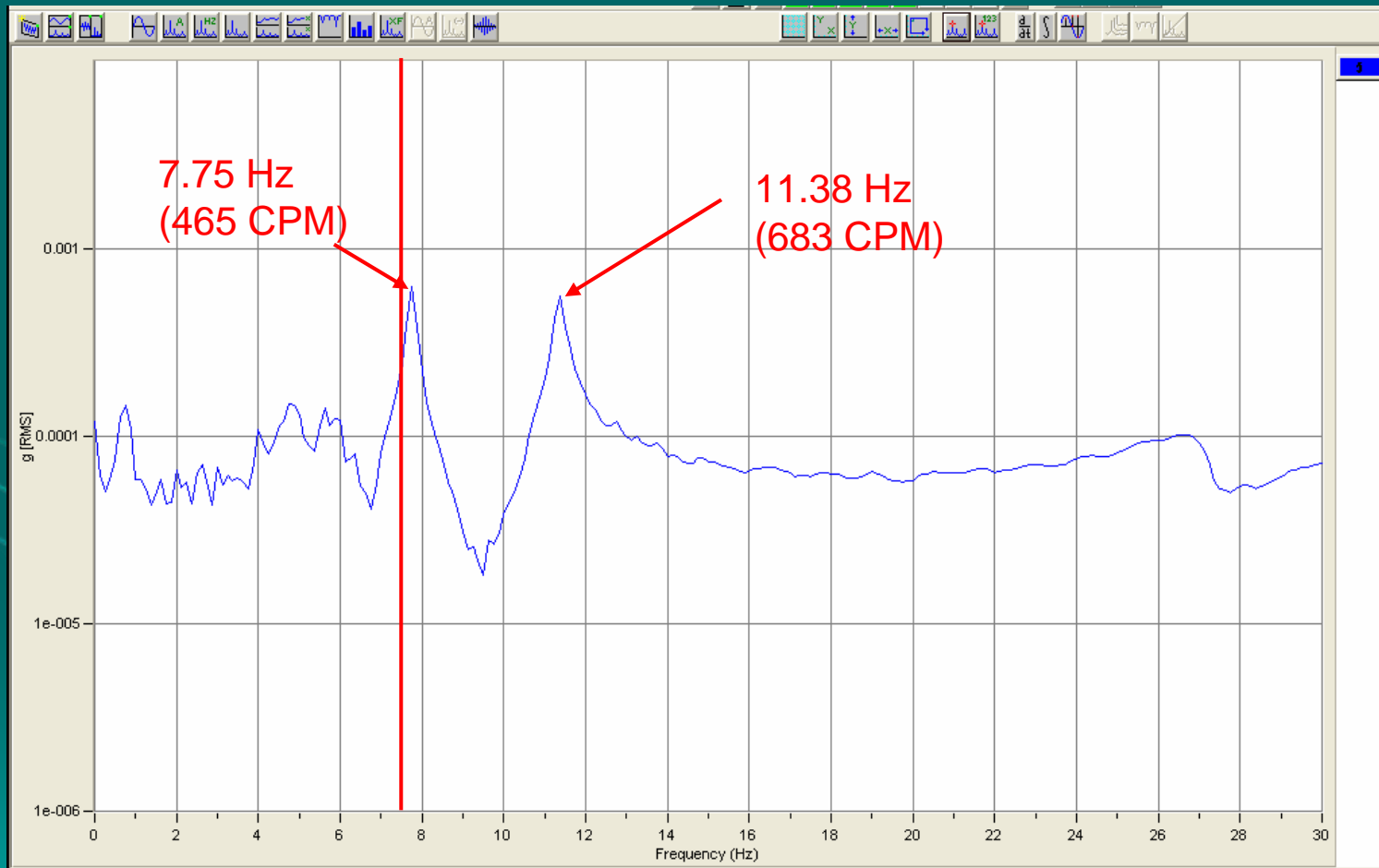


# Field Impact Test Results (parallel-to-discharge)

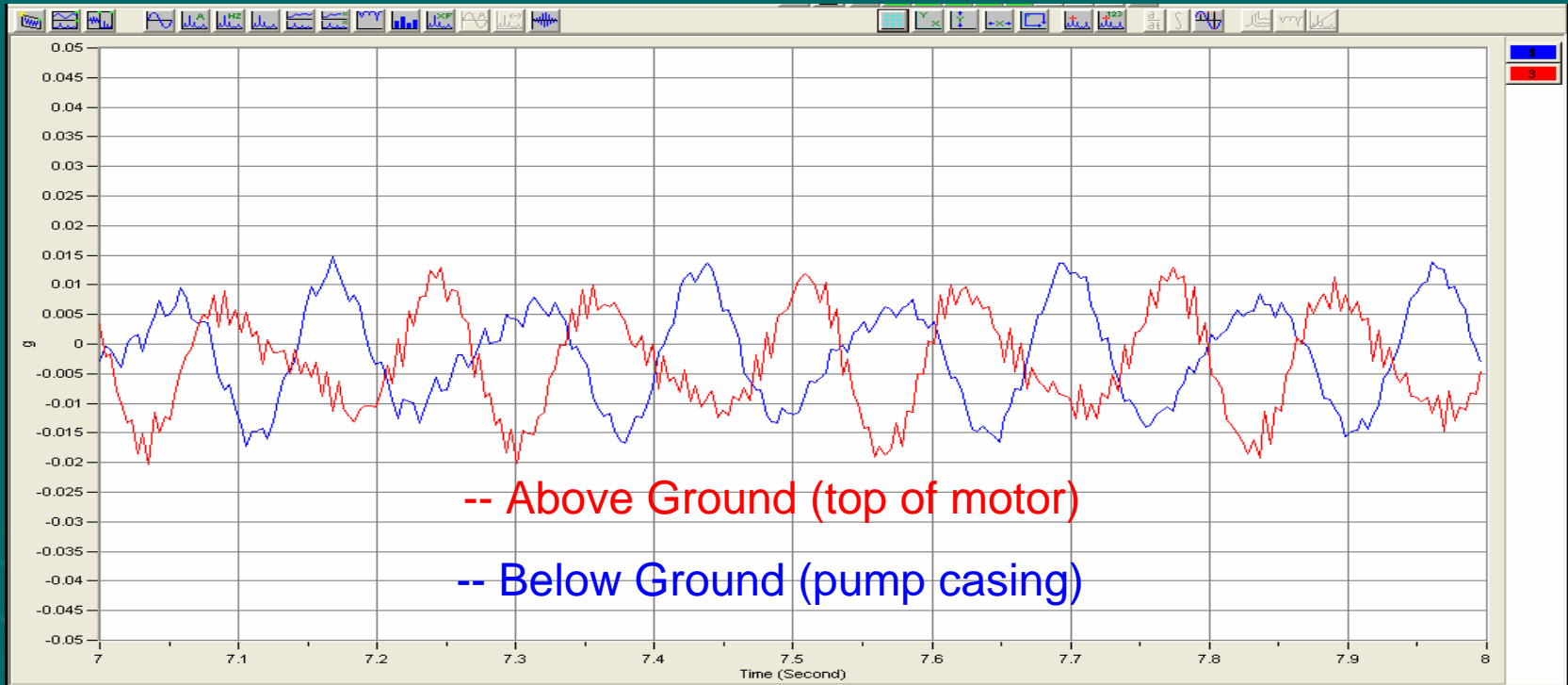




# Field Impact Test Results (perpendicular-to-discharge)



# Field No-Load Motor Test (uncoupled from pump)



- Traces indicate out-of-phase motion of top of motor compared to pump casing
- 1st pump bending mode

# Field Natural Frequencies (semi-wet)

Running speed = 7.42 Hz (445 RPM)		
A Pump		
Direction	Natural Frequency (Hz)	Separation Margin
Parallel (1st Mode)	6.88	-7.3%
Perpendicular (1st Mode)	7.75	<b>4.4%</b>
B Pump		
Direction	Natural Frequency (Hz)	Separation Margin
Parallel (1st Mode)	6.75	-9.0%
Perpendicular (1st Mode)	7.50	<b>1.1%</b>

Note: A pump vibrations were more than 5 times those on the B pump, indicating that the operating natural frequencies shift down by approximately 0.25 Hz.

# Fixing the Problem

- Finite Element Analysis

- Adjusted model to match field measurements, “Model Calibration”
  - Infinitely stiff foundation
  - Doubled motor reed critical frequency
- Analyzed corrective action options with FEA to assure success

- Cut Slots in Stiffening Ribs to De-Tune Natural Frequencies

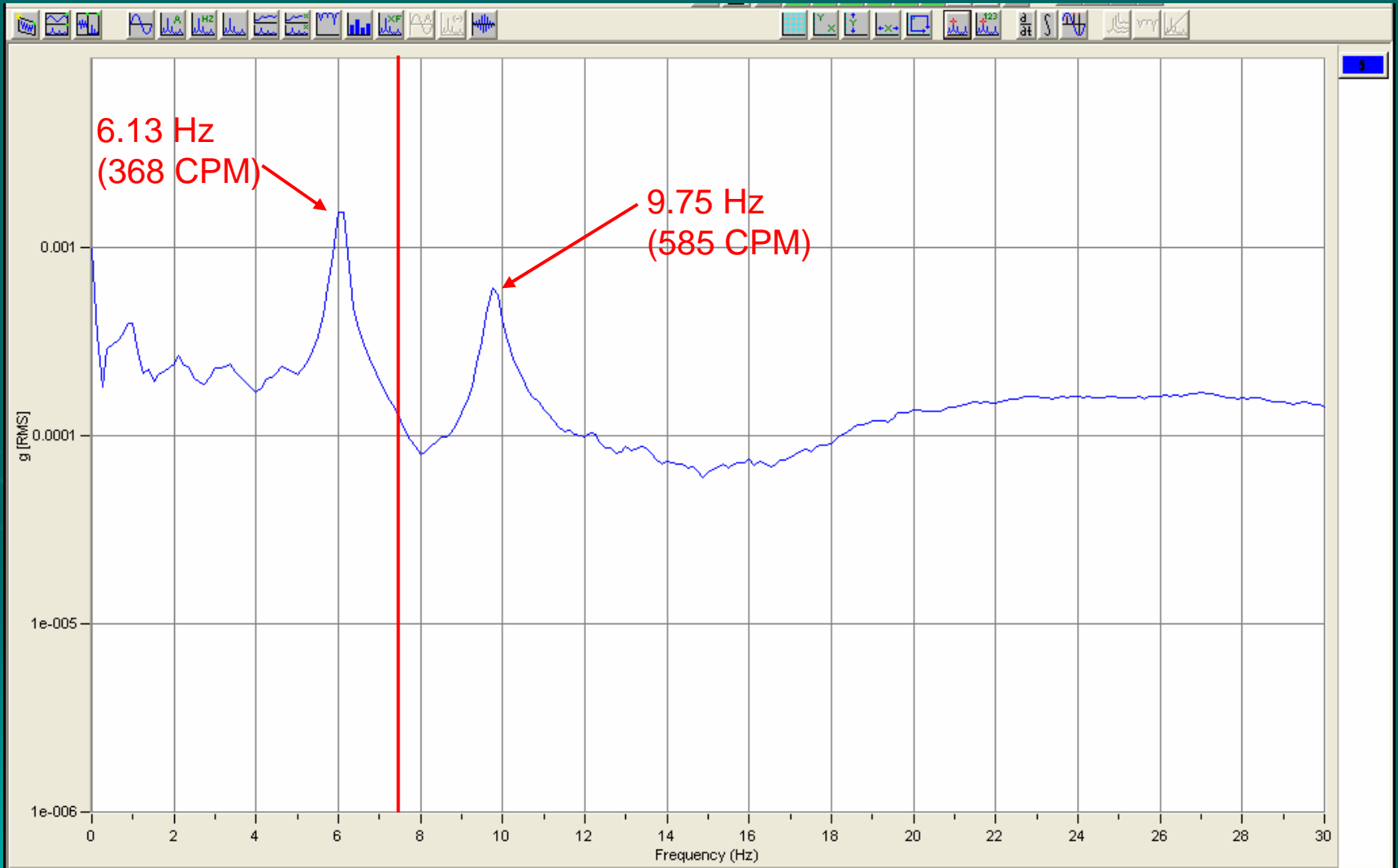
- FEA predicted a minimum separation margin of 15% for the first mode (perpendicular)
- Achieved as-measured separation margin of 12.4% first mode (perpendicular).





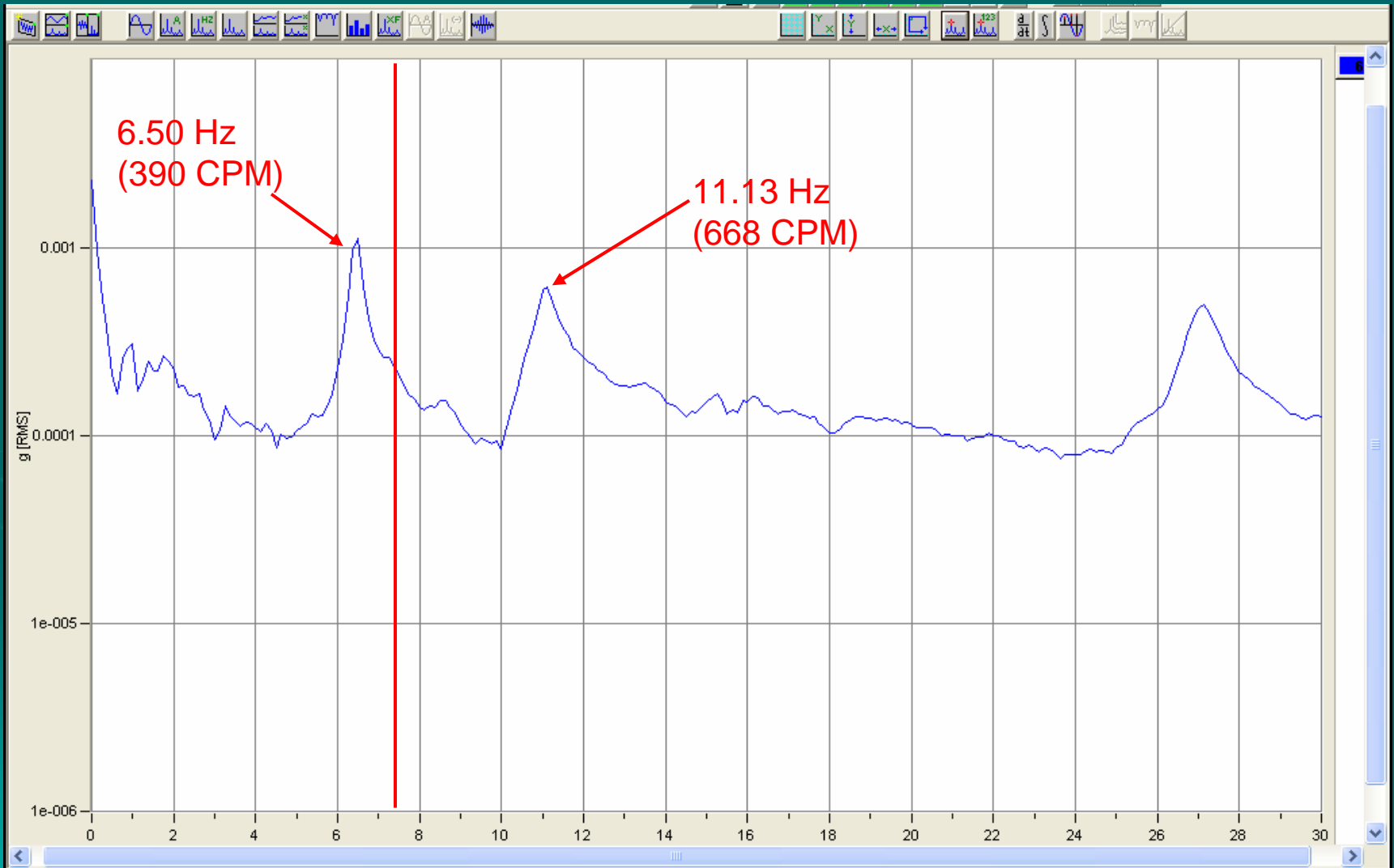


# Modified Pump Impact Test Results (parallel-to-discharge)





# Modified Pump Impact Test Results (perpendicular-to-discharge)




# Final Natural Frequencies

<b>A Pump</b>	<b>Natural frequency (Hz)</b>		<b>% Change</b>	<b>% Separation</b>
	<b>Before</b>	<b>After</b>		
In Line (1st Mode)	6.88	6.13	10.90%	18.27%
Perpendicular (1st Mode)	7.75	6.50	16.13%	13.33%

<b>B Pump</b>	<b>Natural frequency (Hz)</b>		<b>% Change</b>	<b>% Separation</b>
	<b>Before</b>	<b>After</b>		
In Line (1st Mode)	6.75	6.13	9.19%	18.27%
Perpendicular (1st Mode)	7.50	6.50	13.33%	13.33%

Operational vibrations were 0.12 in/s RMS or less on both pumps in both directions and were acceptable.

# Final Vibration Amplitudes

- Operational vibrations less than 0.12 in/s RMS on both pumps in all direction.
  - Vibration meets acceptance standards.
  - Pumps accepted by customer.
- 

# Conclusions

- Factory testing results cannot always ensure acceptable field vibration levels.
- FEA combined with field vibration measurements is an excellent tool for properly diagnosing and correcting unexpected field vibration problems.
- Without field vibration measurement, FEA is only as good as its input data assumptions
  - Foundation information
  - Motor reed critical frequency